

ROTARY PISTON VACUUM PUMP WITH WASHING INSTALLATION

This invention relates to the field of vacuum pumps. In particular, but not strictly limited to vacuum pumps with a screw type configuration.

Screw pumps usually comprise two spaced parallel shafts each carrying

- › externally threaded rotors, the shafts being mounted in a pump housing such that the threads of the rotors intermesh. Close tolerances between the rotor threads at the points of intermeshing and with the internal surface of the pump body, which typically acts as a stator, causes volumes of gas being pumped between an inlet and an outlet to be trapped between the threads of the rotors
- 10 and the internal surface and thereby urged through the pump as the rotors rotate.

Screw pumps are widely regarded as a reliable means for generating vacuum conditions in a multitude of processes. Consequently, they are being applied to an increasing number of industrial processes. Such applications may

- 15 involve materials that have "waxy" or "fatty" properties e.g. tallow based plasticisers. In operation of the pump, these products form deposits on the surfaces of the pump. On shutdown of the pump these surfaces cool, the deposits also cool and solidify within the pump. Where such deposits are located in clearance regions between components, they can cause the pump
- 20 to seize up such that restart is inhibited or even prevented.

Similar problems can be encountered in a number of semiconductor

processes that use vacuum pumps, especially those in the chemical vapour deposition (CVD) category. Such processes can produce a significant amount of by-product material. This can be in the form of powder or dust,

- 25 which may remain loose or become compacted, or in the form of hard solids, especially if the process gas is condensable and sublimes on lower temperature surfaces. This material can be formed in the process chamber, in the foreline between the chamber and the pump, and/or in the vacuum pump itself. If such material accumulates on the internal surfaces of the
- 30 pump during its operation, this can effectively fill the vacant running clearance between the rotor and stator elements on the pump, and can also cause

spikes in the current demand on the motor of the vacuum pump. If this continues unabated, then this build-up of solid material can eventually cause the motor to become overloaded, and thus cause the control system to shut down the vacuum pump. Should the pump be allowed to cool down to 5 ambient temperature, then this accumulated material will become compressed between the rotor and stator elements. Due to the relatively large surface area of potential contact that this creates between the rotor and stator elements, such compression of by-product material can increase the frictional forces opposing rotation by an order of magnitude.

10 In order to release the rotors in prior art pumps, a facility is provided whereby a bar can be inserted into sockets attached to the primary shaft of the rotor through an access panel. This bar is used as a lever to try to rotate the shaft and release the mechanism such that the machine can be restarted. This levering system allows more rotational force to be applied to the internal 15 components than could be exerted by the motor. Such force will be transmitted to the rotor vanes and the associated stresses may prove to be detrimental to the structure of the rotor. If this system fails to release the mechanism it is then necessary to disassemble the apparatus such that a liquid solvent can be poured into the pump casing to dissolve the residue to a 20 level where the shaft can be rotated manually. This disassembly not only causes the pump to be off line for a certain length of time, but it then must be re-commissioned and re-tested to ensure the reliability of the connections to the surrounding apparatus.

It is an aim of the present invention to overcome the aforementioned problems 25 associated with pump technology.

The present invention provides a pump comprising a rotor element and a stator element; a housing enclosing the elements and having an inlet for receiving pumped fluid, and downstream from the inlet, at least one port; and 30 means for injecting, into the housing via said at least one port, fluid for acting on deposits located on the element surfaces to enable said deposits to be removed therefrom. As the port(s) are located downstream of the inlet, any

fluid injected on the rotor and stator elements can be directly injected into the swept volume to impinge on the surfaces of these elements. This can significantly improve cleaning efficiency in comparison to a system where the cleaning fluid is introduced via the housing inlet for pumped fluids. Where 5 many ports are provided, these may be located in an array. For example, the ports may be located radially about the housing, and/or may be located along the length of the rotor element.

The housing may comprise an inner layer and an outer layer between which a cavity may be formed. In operation of the pump a liquid may be passed 10 through this cavity. The inner layer of the housing may act as the stator of the pump.

The port may include a nozzle through which, in use, fluid is sprayed, this nozzle may be integrally formed within the port.

The pump may be a screw pump comprising two threaded rotors in which 15 case the port(s) may be located after the first two complete turns of thread of the rotors from the inlet end of the rotor. Alternatively the pump may be a Northeby ("claw") pump or a Roots pump.

The fluid may be a liquid or a vapour. The fluid may be a solvent for 20 dissolving residue collected on the rotor when the pump is in use or it may be steam. The fluid may comprise a reactive substance for reacting with the deposits, and may comprise, for example, a halogen. Such fluid can be particularly useful as a cleaning fluid when the pump is used as part of a CVD process to remove solid by-products of the CVD process.

25 Thus, the present invention also provides a pump comprising a rotor element and a stator element; a housing enclosing the elements and having at least one port; and means for injecting, into the housing via said at least one port, a fluid comprising a reactive substance for reacting with particulates located on 30 the element surfaces to enable said particulates to be removed therefrom.

The fluid may comprise a halogen, for example fluorine, and may be a fluorinated gas, such as a perfluorinated gas. Examples of such fluid include ClF_3 , F_2 , and NF_3 .

The invention thus extends to chemical vapour deposition apparatus
5 comprising a process chamber and a pump according to any preceding claim
for evacuating the process chamber, wherein, in use, the deposits are a by-
product of a chemical vapour deposition process.

According to the present invention there is further provided a method of
managing deposits within a pump, the pump comprising a rotor element and a
10 stator element, and a housing enclosing the elements and having an inlet for
receiving pumped fluid, and downstream from the inlet, at least one port, the
method comprising injecting, into the housing via said at least one port, fluid
for acting on deposits located on the element surfaces to enable said deposits
to be removed therefrom.

15 The present invention also provides a method for managing deposits within a
pump, the pump comprising a rotor element and a stator element, and a
housing enclosing the elements and having at least one port; the method
comprising injecting, into the housing via said at least one port, a fluid
comprising a reactive substance for reacting with particulates located on the
20 element surfaces to enable said particulates to be removed therefrom.

The delivery of fluid may occur at predetermined intervals during operation of
the pump, for example, using solenoid valve control. Furthermore a
monitoring step may be performed wherein the performance of the pump is
monitored, for example, by measuring at least one of the group of rotor speed,
25 power consumption, and volumetric gas flow rate. These measured
parameters may be used to determine the extent of accumulation of deposits
on the internal working surfaces of the pump. A fluid flow rate may then be
calculated, this rate being that of the delivered fluid that would be sufficient to
compensate for the quantity of accumulated deposits as determined above.
30 Subsequently, the flow rate of fluid being delivered to the rotor may be
adjusted to reflect the new calculated value.

According to the present invention there is further provided a method for managing deposits within a pump mechanism by introducing fluid suitable for dissolving, diluting or otherwise disengaging deposits which have accumulated on the internal working surfaces of the pump, the method

5 comprising the steps of:

- (a) monitoring the performance of the pump, for example, by recording at least one of the group of rotor speed, power consumption, and volumetric gas flow rate;
- (b) calculating the rate of accumulation of deposits on the internal working surfaces of the pump based on the monitored performance;
- 10 (c) calculating a fluid flow rate required to compensate for the accumulation of deposits as determined in step (b); and
- (d) effecting an adjustment of the flow rate of fluid being delivered to the rotor to reflect the calculated value from step (c).

15 The pump may be inoperative as the fluid is delivered, for example where seizure has occurred or where cleaning needs to take place. In this case, the method may further involve applying torque to the rotors of the pump in order to overcome any remaining impeding force potentially caused by deposits located on the internal working components of the pump. Under certain 20 conditions, for example where the material being transported is particularly viscous or waxy and this viscosity may reduce with an increase in temperature, the method may further involve the introduction of thermal fluid into a cavity provided within the housing of the pump, where this cavity encircles the rotor components. This thermal fluid may be heated in order to 25 raise the temperature of the fluid and the deposits sufficiently to release the deposits prior to applying the torque as discussed above.

The controller of the dry pump apparatus may comprise a microprocessor which may be embodied in a computer, which in turn is optionally programmed by computer software which, when installed on the computer, 30 causes it to perform the method steps (a) to (d) mentioned above. The carrier medium of this program may be selected from but is not strictly limited to a floppy disk, a CD, a mini-disc or digital tape.

An example of the present invention will now be described with reference to the accompanying drawings in which:

Figure 1 illustrates a schematic of a screw pump of the present invention;

5

Figure 2 illustrates a schematic of a double-ended screw pump of the present invention;

Figure 3 is an end sectional view of the pump of Figures 1 and 2;

10

Figure 4 is a detailed view of a section of a water jacket that illustrates the implementation of an injection port; and

Figure 5 illustrates an arrangement for supplying fluid to a pump

Whilst the example pumps illustrated in Figures 1 and 2 are screw pumps it is

15

envisioned that this invention can be applied to any type of vacuum pump, in particular claw pumps.

In the example of Figure 1, two rotors 1 are provided within an outer housing 5 that serves as the stator of the pump. The two contra-rotating, intermeshing rotors 1 are positioned such that their central axes lie parallel to one another.

20

The rotors are mounted through bearings 10 and driven by a motor 11 (shown in Figure 2). Injection ports 2 are provided along the length of the rotor, in the examples of Figures 1 and 2 (shown as solid lines in Figure 3) these ports 2 are located laterally within the pump on the opposite side of the rotors from the intermeshing region of the rotors. However, the ports may be positioned at any radial location around the stator 5. Some of these locations are illustrated in Figure 3.

30

The ports 2, which may contain nozzles to allow the fluid to be sprayed, are preferably distributed along the length of the stator component 5 such that the solvent or steam can be easily applied over the entire rotor. Alternatively, this distribution of ports allows the fluid to be readily concentrated in any particular problem area that may arise. This is especially important when solvent is

injected during operation, in order to limit the impact on pump performance. If, for example, a single port was to be used at the inlet 3 of the pump, this may have a detrimental effect on the capacity of by-products that could be transported away from the evacuated chamber (not shown) by the pump. By 5 bringing solvent into contact with the rotor 1 after the first few turns of the thread, the likelihood of backward contamination of the solvent into the chamber will be reduced.

Furthermore, where solvent is introduced in the inlet region of the pump, the pressure is such at the inlet that there is an increased risk that the solvent will 10 flash. In processes where it is necessary for the solvent to remain in liquid phase the solvent must be introduced closer towards the exhaust region of the pump where the pressures will have risen. As solvent is introduced through a number of ports 2 along the length of the stator, the overall effect is to gradually increase the quantity of solvent present, as the likelihood of residue 15 build up on the rotor 1 increases towards the exhaust stages. An additional benefit may be seen in some configurations where addition of liquid into the final turns of thread of the rotor will act to seal the clearances between the rotor and the stator in this region of the pump. Thus leakage of gas will be substantially reduced and performance of the pump will be improved.

20 In some processes, it is not appropriate to introduce solvent during operation as the waste products from the evacuated chamber are collected at the outlet of the pump for a particular purpose and this material ought not to be contaminated. Other applications may not result in levels of residue that warrant constant injection of solvent during operation. In these cases, and 25 where an unplanned shut down of the pump occurs such that standard practices, such as purging, are not followed, the residue from the process cools down as the apparatus drops in temperature. In these circumstances a seizure of the mechanism may occur as deposits build up and become more viscous or solidify. In a system according to the present invention, the 30 injection ports 2 can be used to introduce a solvent into the stator cavity 6 in a distributed manner without needing to go to the expense or inconvenience of disassembling the apparatus. Once the solvent has acted upon the deposits

to either soften or dissolve them, the shaft may then be rotated either by using the motor or manually to release the components without applying excessive, potentially damaging, force to the rotor.

Delivery of fluid may be performed through simple ports as liquid is drip-fed

5 through a hole in the housing or nozzles may be provided through which the fluid may be sprayed. Control systems may be introduced such that the solvent delivery can be performed in reaction to the changing conditions being experienced within the confines of the pump apparatus. For example, in the arrangement shown in Figure 5, a control system 20 supplies cleaning fluid, 10 for example, stage by stage, to the ports 2 of pump 21 via supply conduits 22. As indicated at 24, a purge gas system may also be provided for supplying a purge gas, such as nitrogen to the pump 21.

Where the process material is waxy or fatty, compatible solvents will need to

15 be introduced to perform the dilution/cleaning function. Such solvents may be provided in liquid or vapour form. Any compatible, effective cleaning medium may be used such as xylene in the case of hydrocarbon based/soluble products or water in the case of aqueous based / soluble products, alternatively, detergents may be used.

20 Where the process material is a by-product of a CVD process, the cleaning fluid may comprise a fluorinated gas. Examples of such cleaning fluid include, but are not restricted to, ClF₃, F₂, and NF₃. The high reactivity of fluorine means that such gases would react with the solid by-products on the 25 pump mechanism, in order to allow the by-products to be subsequently flushed from the pump with the exhausted gases. To avoid corrosion of internal components of the pump by the fluorinated gases, materials need to be carefully selected for use in forming components of the pump, such as the rotor and stator elements, and any elastomeric seals, which would come into 30 contact with the cleaning gas.

The housing 5 as illustrated in Figure 3 is provided as a two-layer skin construction, an inner layer 6 and an outer layer 9. It is the inner layer 6 that

acts as the stator of the pump. A cavity 7 is provided between the layers 6, 9 of the housing 5 such that a cooling fluid, such as water, can be circulated around the stator in order to conduct heat away from the working section of the pump. This cavity 7 is provided over the entire length of the rotor i.e. over 5 the inlet region 3 as well as the exhaust region 4. Under circumstances where the pump has become seized due to cooling of the rotor which, in turn, solidifies residues on the surfaces between the rotor and the stator, the 'cooling liquid' in the cavity 7 of the housing 5 may be heated to raise the temperature of the rotor 1. This can enhance the pliability of the residue and 10 may assist in releasing the mechanism. The housing 5 is provided with pillars 8 of solid material through the cavity 7 in order to provide regions where injection ports 2 can be formed.

The present invention is not restricted for use in screw pumps and may readily be applied to other types of pump such as Northe (‘claw’) pumps or Roots 15 pumps.

In summary, a pump comprises at least one rotor 1, a stator 5 and a housing 5, the rotor 1 being enclosed by the housing 5. The housing 5 comprises at least one port 2 extending through the housing 5 to enable delivery of a fluid directly onto a surface of the at least one rotor 1.

20 It is to be understood that the foregoing represents just a few embodiments of the invention, others of which will no doubt occur to the skilled addressee without departing from the true scope of the invention as defined by the claims appended hereto.